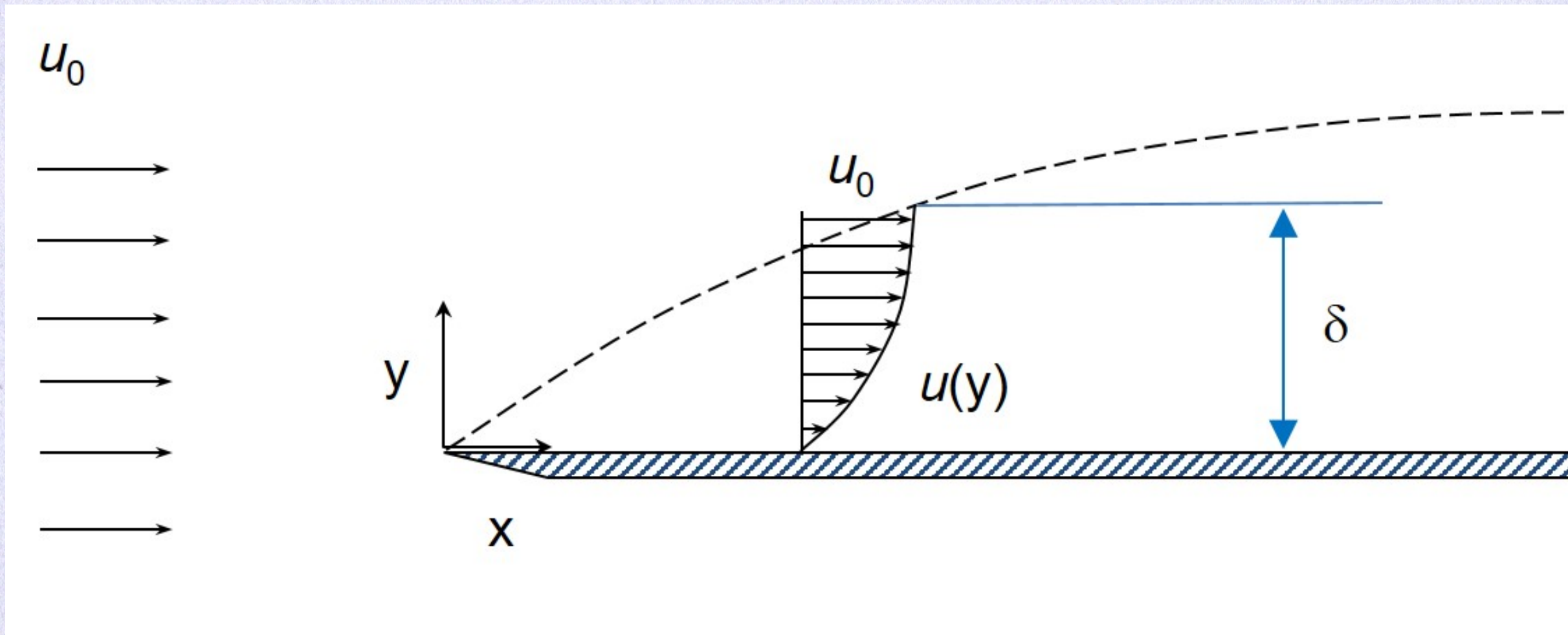


Repaso de Clase 7

- Fricción entre elementos de fluido >>> Viscosidad de Newton >>> $\tau = \frac{F}{A} = \mu \frac{\partial u_x}{\partial y}$
- Tensor de esfuerzos viscosos >>> $\sigma'_{i,j} = \mu (\partial_i u_j + \partial_j u_i - \frac{2}{3} \partial_k u_k \delta_{i,j}) + \mu' \partial_k u_k \delta_{i,j}$
- Ecuación de Navier-Stokes >>> $\rho(\partial_t \underline{u} + (\underline{u} \cdot \underline{\nabla}) \underline{u}) = \rho \underline{f} - \underline{\nabla} p + \mu \nabla^2 \underline{u} + (\frac{\mu}{3} + \mu') \underline{\nabla}(\underline{\nabla} \cdot \underline{u})$
- Condiciones de contorno para fluidos viscosos >>> rigidez (fluido ideal) + no deslizamiento
- Viscosidad >>> Disipación de energía ($\frac{dE}{dt} \leq 0$)
- Número de Reynolds >>> $Re = \frac{u_0 L_0}{\nu}$

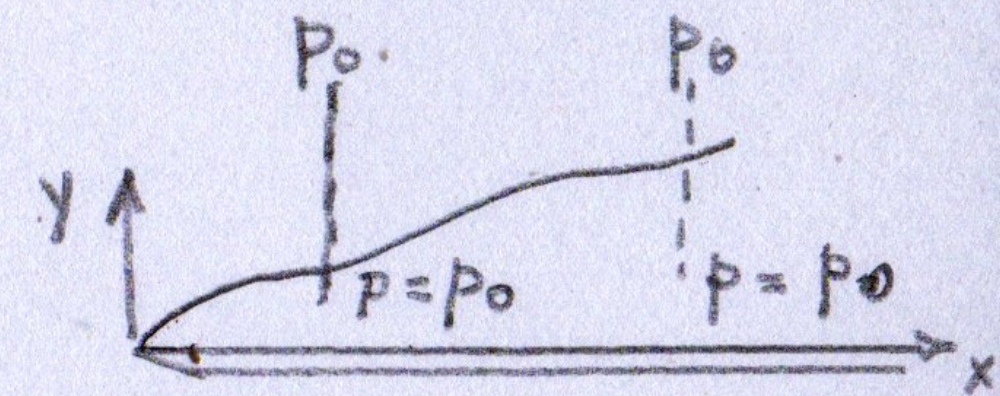
Capa límite



$$\hat{x}) \quad (u_0 + \delta u) \frac{\partial}{\partial x} (u_0 + \delta u) = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \quad \begin{matrix} \delta u \ll u_0 \\ \frac{\partial^2 u}{\partial x^2} \ll \frac{\partial^2 u}{\partial y^2} \end{matrix}$$

$$\hat{y}) \quad 0 = -\frac{1}{\rho} \frac{\partial p}{\partial y} \rightarrow p = p(x)$$

Pero $p(x, y) = p(x, \infty) = p_0$



$$\therefore \frac{\partial p}{\partial x} = 0 \rightarrow u_0 \frac{\partial \delta u}{\partial x} \approx \nu \frac{\partial^2 \delta u}{\partial y^2}$$

Ecuación de difusión $\rightarrow$ Cond. Contorno $\begin{cases} \delta u(y \rightarrow \infty) = 0 \\ \delta u(y=0) = -u_0 \end{cases}$

Buscamos soluciones auto-similares para esta ecuación, donde la incógnita δu dependa solo de

$$\xi = \frac{y}{\sqrt{4\nu x/u_0}} \quad \delta u(x, y) = \delta u(\xi(x, y)) \quad \begin{cases} \frac{\partial \delta u}{\partial x} = \frac{\partial \xi}{\partial x} \frac{d\delta u}{d\xi} \\ \frac{\partial \delta u}{\partial y} = \frac{\partial \xi}{\partial y} \frac{d\delta u}{d\xi} \end{cases}$$

Entonces:

$$\frac{\partial \delta u}{\partial x} = -\frac{\xi}{2x} \delta u'$$

$$\frac{\partial \delta u}{\partial y} = \frac{\xi}{y} \delta u' \rightarrow \frac{\partial^2 \delta u}{\partial y^2} = \frac{\xi^2}{y^2} \delta u''$$

Las condiciones de contorno de un fluido ideal o viscoso, son marcadamente diferentes. Se espera que las respectivas soluciones difieran apreciablemente cerca de los contornos $\rightarrow$ capas límite

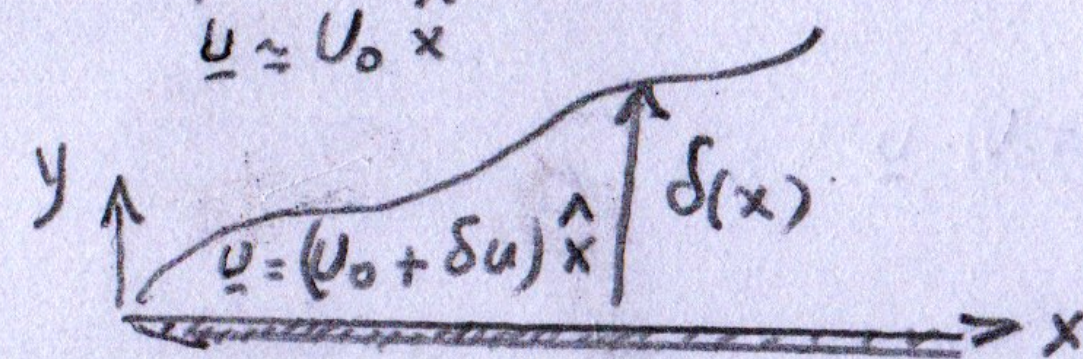
En el problema de la figura suponemos:

- plano: $\underline{u} = U_x(x, y) \hat{x}$

- estacionario: $\frac{\partial}{\partial t} = 0$

- incompresible: $\rho = \text{cte}$

- lineal: $Re \ll 1 \rightarrow U_x \approx U_0 + \delta u, \delta u \ll U_0$



Capa límite

Reemplazando: $\delta u'' = -2\xi \delta u'$

La primera integral: $\delta u' = A e^{-\xi^2}$

La segunda integral involucra a la llamada función error definida como:

$$\text{ferr}(\xi) = \frac{2}{\sqrt{\pi}} \int_0^{\xi} d\xi e^{-\xi^2}$$

tal que $\text{ferr}(0) = 0$ y $\text{ferr}(\infty) = 1$

$$\delta u = A \int e^{-\xi^2} d\xi = C + \frac{\sqrt{\pi} A}{2} \text{ferr}(\xi)$$

Con las condiciones de contorno, fijamos los valores de las constantes de integración:

$$\bullet \delta u(\xi \rightarrow \infty) = 0 \longrightarrow C + \frac{\sqrt{\pi} A}{2} = 0$$

$$\bullet \delta u(\xi=0) = -U_0 \longrightarrow C = -U_0$$

$$\therefore u_x(x,y) = U_0 + \delta u(x,y) = U_0 \text{ferr}\left(\frac{y}{\sqrt{4\nu x/U_0}}\right)$$

Para interpretar este resultado, definimos el espesor de la capa límite

$$\delta(x) = \sqrt{\frac{2\nu x}{U_0}}$$

de modo que

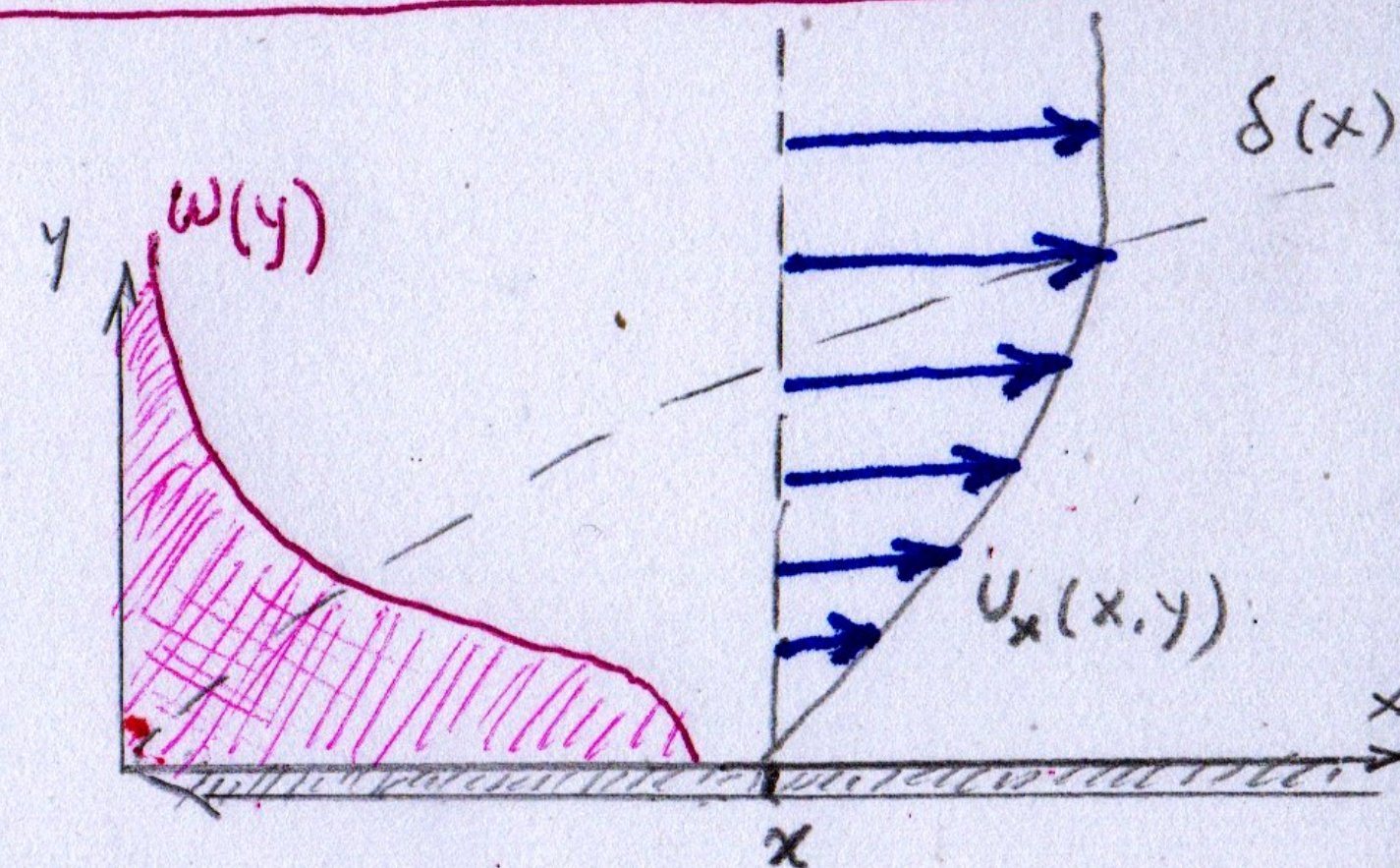
$$u_x(x,y) = U_0 \text{ferr}\left(\frac{y}{\sqrt{2} \delta(x)}\right)$$

Calculamos la vorticidad de este flujo:

$$\underline{\omega} = \nabla \times \underline{u} \longrightarrow \omega_z = -\frac{\partial u_x}{\partial y}$$

$$\text{ferr}'(\xi) = \frac{2}{\sqrt{\pi}} e^{-\xi^2} \longrightarrow \omega_z = -\sqrt{\frac{2}{\pi}} \frac{U_0}{\delta(x)} e^{-\frac{y^2}{2\delta(x)^2}}$$

Es decir que en una dada posición x , la vorticidad se distribuye en altura con un perfil gaussiano.



Ley de Stokes

Calculemos la fuerza que ejerce un flujo estacionario, laminar e incompresible sobre una esfera

$$\rho \left(\underbrace{\frac{\partial \underline{u}}{\partial t}}_{=0 \text{ (estac.)}} + \underbrace{(\underline{u} \cdot \nabla) \underline{u}}_{=0 \text{ (Re} \ll 1)} \right) = -\nabla \beta + \mu \nabla^2 \underline{u}$$

Por lo tanto;

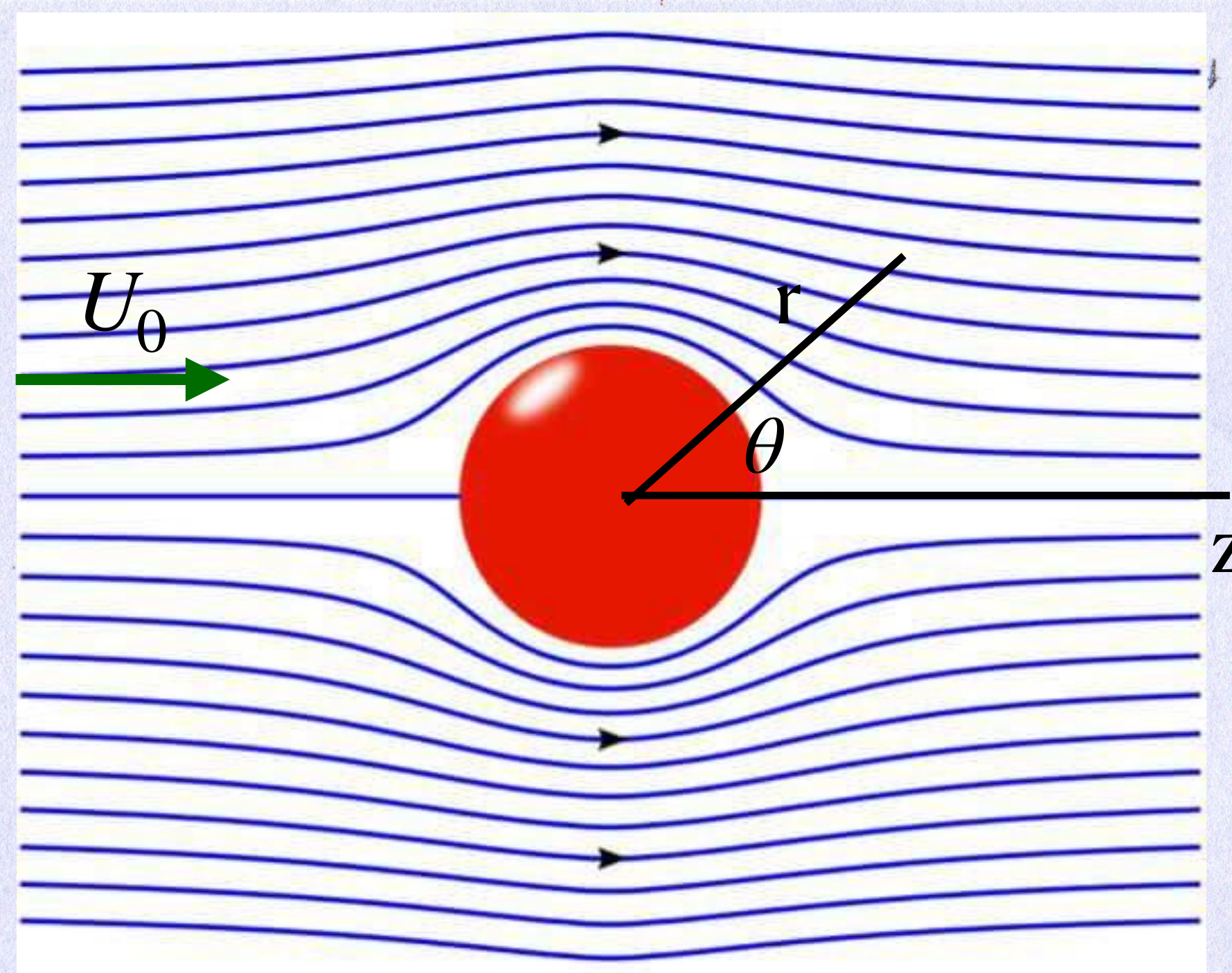
$$\boxed{\begin{aligned} \nabla \beta &= \mu \nabla^2 \underline{u} \\ \nabla \cdot \underline{u} &= 0 \quad \underline{u}(r=a) = 0 \end{aligned}}$$

El problema es axisimétrico. En esféricas:

$$\underline{u} = u_r(r, \theta) \hat{r} + u_\theta(r, \theta) \hat{\theta} \quad \frac{\partial}{\partial \phi} = 0$$

$$\nabla \cdot \underline{u} = 0 \longrightarrow \underline{u} = \nabla \times (\psi \hat{\phi})$$

$$u_r = \frac{1}{r^2 \sin \theta} \frac{\partial \psi}{\partial \theta} \quad u_\theta = -\frac{1}{r \sin \theta} \frac{\partial \psi}{\partial r}$$



La vorticidad resulta:

$$\underline{\omega} = \nabla \times \underline{u} = -\frac{\hat{\phi}}{r \sin \theta} E^2 \psi$$

donde $E^2 = \frac{\partial^2}{\partial r^2} + \frac{\sin \theta}{r^2} \frac{\partial}{\partial \theta} \left(\frac{1}{\sin \theta} \frac{\partial}{\partial \theta} \right)$

La ecuación N-S resulta:

$$\textcircled{r} \quad \frac{\partial \beta}{\partial r} = \frac{\mu}{r^2 \sin \theta} \frac{\partial}{\partial \theta} (E^2 \psi)$$

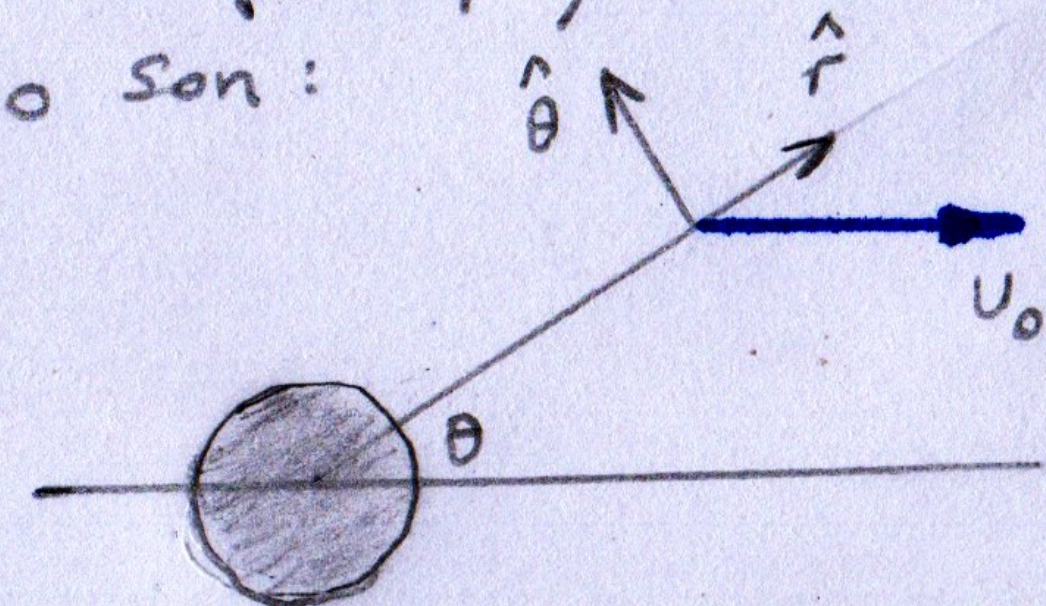
$$\textcircled{\theta} \quad \frac{1}{r} \frac{\partial \beta}{\partial \theta} = -\frac{\mu}{r \sin \theta} \frac{\partial}{\partial r} (E^2 \psi)$$

Como $\frac{\partial^2}{\partial r^2} \beta = \frac{\partial^2}{\partial \theta^2} \beta \longrightarrow E^2(E^2 \psi) = 0$

Las condiciones de contorno son:

$$\bullet r=a \longrightarrow \frac{\partial \psi}{\partial r} = 0 = \frac{1}{r} \frac{\partial \psi}{\partial \theta}$$

$$\bullet r=\infty \longrightarrow \begin{aligned} u_r &= u_0 \cos \theta \\ u_\theta &= -u_0 \sin \theta \end{aligned}$$



Ley de Stokes

$$r=\infty \rightarrow \left\{ \begin{aligned} u_r &= u_0 \cos \theta = \frac{1}{r^2 \sin \theta} \partial_\theta \psi \\ u_\theta &= -u_0 \sin \theta = -\frac{1}{r \sin \theta} \partial_r \psi \end{aligned} \right\} \rightarrow \psi_{(r \rightarrow \infty)} = \frac{u_0 r^2 \sin^2 \theta}{2}$$

Proponemos soluciones de la forma:

$$\psi(r, \theta) = f(r) \sin^2 \theta \xrightarrow{E^2(E^2 \psi) = 0} \left(\partial_{rr}^2 - \frac{2}{r^2} \right) f = 0$$

$$\text{Probamos } f(r) \sim r^\alpha \rightarrow [(\alpha-2)(\alpha-3)-2][\alpha(\alpha-1)-2] = 0$$

$$\text{Resulta } \alpha = -1, 1, 2, 4$$

$$\text{Por lo tanto } f(r) = \frac{A}{r} + Br + Cr^2 + Dr^4$$

$$r=\infty \rightarrow \psi \rightarrow \frac{u_0 r^2 \sin^2 \theta}{2} \begin{cases} D=0 \\ C = \frac{u_0}{2} \end{cases}$$

$$r=a \rightarrow \partial_r \psi|_{r=a} = 0 = \left(-\frac{A}{a^2} + B + u_0 a \right) \sin^2 \theta$$

$$\rightarrow \partial_\theta \psi|_{r=a} = 0 = \left(\frac{A}{a} + Ba + \frac{u_0}{2} a^2 \right) 2 \sin \theta \cos \theta$$

$$\left. \begin{aligned} A &= \frac{u_0 a^3}{4} \\ B &= -\frac{3}{4} u_0 a \end{aligned} \right\} \rightarrow \psi(r, \theta) = \frac{u_0 \sin^2 \theta}{4} \left(2r^2 + \frac{a^3}{r} - 3ar \right)$$

De cualquiera de las componentes de N-S integramos para obtener la presión:

$$p(r, \theta) = p_0 - \frac{3}{2} \frac{\mu u_0 a}{r^2} \cos \theta$$

Finalmente, para calcular la fuerza sobre la esfera:

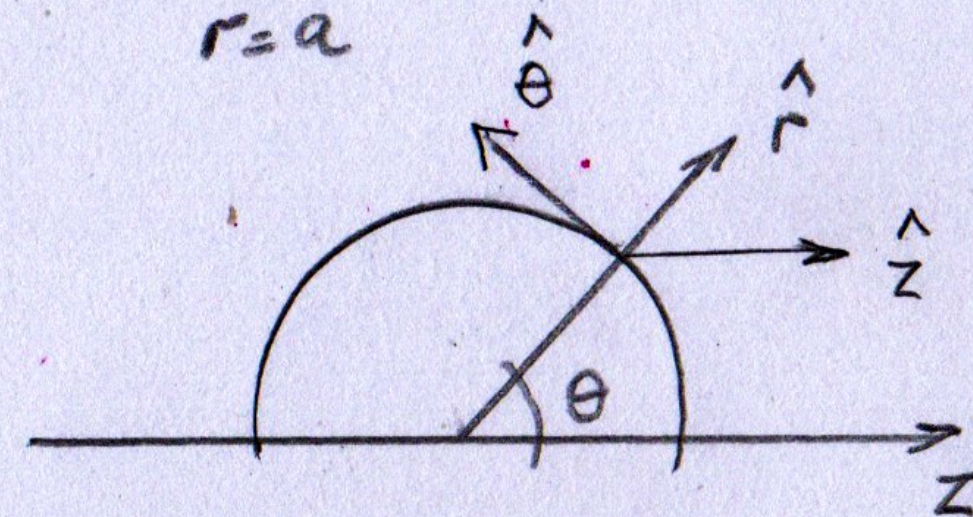
$$\underline{F} = \oint_{r=a} \underline{\sigma} \cdot \hat{r} dS \xrightarrow{\partial_\phi = 0} F_z = \oint_{r=a} \hat{z} \cdot \underline{\sigma} \cdot \hat{r} dS$$

Como:

$$\sigma_{rr} = -p + 2\mu \partial_r u_r$$

$$\sigma_{r\theta} = \mu r \partial_r \left(\frac{u_\theta}{r} \right) + \frac{\mu}{r} \partial_\theta u_r$$

$$\hat{z} \cdot \underline{\sigma} \cdot \hat{r} = \sigma_{rr} \cos \theta - \sigma_{r\theta} \sin \theta$$



Ley de Stokes

Necesitamos calcular

$$u_r = \frac{1}{r^2 \sin \theta} \partial_\theta \psi = \frac{u_0 \cos \theta}{2r^2} (2r^2 + \frac{a^3}{r} - 3ar)$$

$$u_\theta = -\frac{1}{r \sin \theta} \partial_r \psi = -\frac{u_0 \sin \theta}{4r} (4r - \frac{a^3}{r^2} - 3a)$$

y luego:

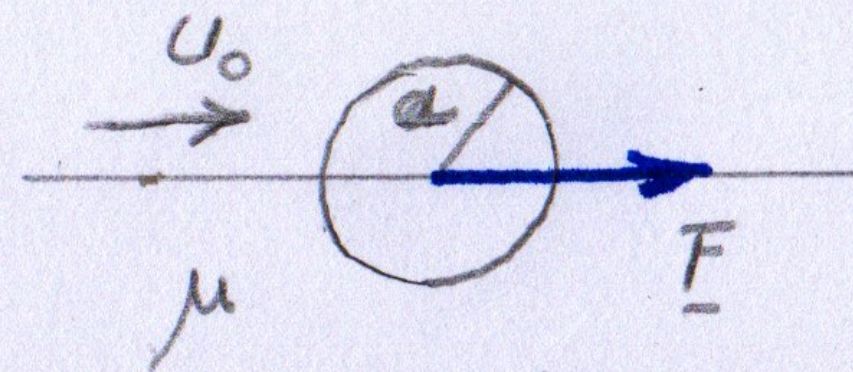
$$\partial_r u_r \Big|_{r=a} = 0$$

$$\partial_\theta u_r \Big|_{r=a} = 0$$

$$\partial_r \left(\frac{u_\theta}{r} \right) \Big|_{r=a} = -\frac{3}{2} \frac{u_0 \sin \theta}{a^2}$$

Finalmente

$$F_z = 6\pi\mu a u_0$$



- Las líneas continuas son líneas de corriente ($\psi = \text{cte}$)
- Las punteadas son de $p = \text{cte}$.
- La paleta de colores muestra vorticidad ω_φ

$$\omega_\varphi = -\frac{3}{2} \frac{u_0 a \sin \theta}{r^2}$$

